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GB 0216258.4

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Incorporated in the United Kingdom,

[ADP No. 08465304001]

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15JUL02 E733110-3 D00001 P01/7700 0.00-0216258.4

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Your reference

GCL/8730

2. Patent application number (The Patent Office will fill in this part)

0216258.4

3. Full name, address and postcode of the or of each applicant (underline all surnames)

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Patents ADP number (if you know it)

8354870001

If the applicant is a corporate body, give the country/state of its incorporation

Title of the invention

A mechanism including a piston-and-cylinder assembly

5. Name of your agent (if you have one)

Abel & Imray

"Address for service" in the United Kingdom to which all correspondence should be sent (including the postcode)

20 Red Lion Street London WC1R 4PQ United Kingdom

Patents ADP number (if you know it)

174001

6. If you are declaring priority from one or more earlier patent applications, give the country and the date of filing of the or of each of these earlier applications and (if you know it) the or each application number

Country

Priority application number (if you know it)

Date of filing (day / month / year)

United Kingdom (GB) 0207390.6

28 March 2002

7. If this application is divided or otherwise derived from an earlier UK application, give the number and the filing date of the earlier application

Number of earlier application

Date of filing (day / month / year)

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No

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A mechanism including a piston-and-cylinder assembly

The invention relates to a mechanism including a piston-and-cylinder assembly.

The invention provides a mechanism including:

a piston-and-cylinder assembly including a piston housed in a cylinder,

a pin member passing through the piston and

a guide member having a guide slot accommodating an

10 end of the pin member,

the guide slot being so shaped and orientated in relation to the piston-and-cylinder assembly that a common axis exists between the guide slot and the piston-and-cylinder assembly,

the guide member and the piston-and-cylinder assembly being so mounted as to be rotatable relative to each other about the common axis and

the guide slot being so shaped as to guide the pin member in a continuous curve, causing the piston to sweep up and down the cylinder, when the guide member and the piston-and-cylinder assembly rotate relative to each other.

Preferably, the mechanism includes axial slots in the cylinder, the pin member engaging the axial slots and, preferably, the axial slots lie on a diameter of the cylinder.

An aspect of the invention is the provision of a mechanism including:

a piston-and-cylinder assembly including a first piston housed in a first cylinder and a second piston housed in a second cylinder,

a first pin member passing through the first piston and a second pin member passing through the second piston and

a guide member having a guide slot accommodating an end of the first pin member and an end of the second pin member,

the guide slot being so shaped and orientated in relation to the piston-and-cylinder assembly that a common axis exists between the guide slot and the piston-and-cylinder assembly,

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being so mounted as to be rotatable relative to each other about the common axis and

the guide slot being so shaped as to guide the pin members in a continuous curve, causing the first and second piston to sweep up and down the cylinder, when the guide member and the piston-and-cylinder assembly rotate relative to each other.

Preferably, the form of the piston-and-cylinder assembly permits the first and second cylinders to lie diametrically opposed to each other.

Preferably, the mechanism includes at least one 20 further pair of diametrically opposed cylinders on the piston-and-cylinder assembly,

further pistons in the further cylinders and further pin members passing through the pistons and being accommodated in the guide slot.

Preferably, the guide slot is of a rhomboid shape with curved corners and, preferably, the curved corners are parts of ellipses.

Alternatively, the guide slot is elliptical in shape.

As another alternative, the guide slot includes three parts which are elliptical in shape and a fourth part which is substantially straight, the straight part occupying a position corresponding to the expansion stroke of the piston or pistons.

As yet another alternative, the guide slot has a dumbell shape including curved end portions separated by a narrower waist portion.

Preferably, the mechanism includes a second guide

5 member having a second guide slot accommodating the other
end of the pin member or the other ends of the pin members,
the second guide slot being so shaped and orientated in
relation to the piston-and-cylinder assembly as to share
the common axis existing between the first guide slot and
the piston-and-cylinder assembly

Preferably, the mechanism includes axial slots in the cylinders, the pin members engaging the axial slots and, preferably, the axial slots lie on diameters of the cylinders.

In one arrangement, the common axis is the axis of a shaft on which the piston-and-cylinder assembly is rotatably mounted, the remainder of the mechanism being fixed.

In another arrangement, the common axis is the axis of a shaft on which the guide member is rotatably mounted or the guide members are rotatably mounted, the piston-and-cylinder assembly being fixed.

In a further arrangement, the guide member is or the guide members are rotatably mounted on the shaft and so is the piston-and-cylinder assembly.

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Preferably, the mechanism includes bearing means at the end of the pin member or the ends of pin members accommodated in a guide slot, for effecting rolling contact between the peripheries of the guide slot and the end of the pin member or the ends of the pin members.

Preferably, the bearing means at the end of the pin member or the ends of the pin members includes an outer bearing assembly contacting only the outer periphery of a

guide slot and an inner bearing assembly contacting only the inner periphery of the guide slot.

Preferably, the outer bearing assembly includes an outer cylindrical shell supported by a plurality of outer rollers on the pin member, the outer cylindrical shell lying in contact with the outer periphery only of the guide slot.

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Preferably, the inner bearing assembly includes an inner cylindrical shell supported by a plurality of inner rollers on the pin member, the inner cylindrical shell lying in contact with the inner periphery only of the guide slot.

In one arrangement, the outer and inner bearing assemblies are so mounted that the outer and inner cylindrical shells rotate about the same axis.

In one arrangement, the outer and inner bearing assemblies are so mounted that the outer and inner cylindrical shells rotate about the axis of the pin member.

In an alternative arrangement, the outer and inner bearing assemblies are so mounted that the outer cylindrical shell rotates about an axis which is offset from the axis about which the inner cylindrical shell rotates.

In one arrangement, the bearing means includes a ball bearing between the outer and inner bearing assemblies, the balls of the ball bearing running in tracks in the outer and inner cylindrical shells.

Preferably, the mechanism includes a guide slot having an inner periphery including a step in its profile for accommodating bearing means at the end of the pin member or the ends of the pin members, the bearing means including an outer bearing assembly contacting only the outer periphery of a guide slot and an inner bearing assembly contacting only the inner periphery of the guide slot.

Preferably, the mechanism includes a guide slot having an outer periphery which is narrower than the inner periphery, bearing means at the end of the pin member or the ends of the pin members including an outer bearing assembly contacting only the narrower outer periphery of a guide slot and an inner bearing assembly contacting only the inner periphery of the guide slot.

Preferably, a plurality of apertures are included in the pin member or pin members for receiving and distributing lubricant to the end of the pin member or pin members.

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Preferably, the mechanism includes a guide member having at least one aperture so positioned as to permit the delivery of lubricant through the guide member to the pin member or pin members.

A mechanism in accordance with the invention may be included in a heat engine, wherein the pistons and cylinders are pistons and cylinders of the heat engine and, in operation, generate motive power for the mechanism.

Preferably, the heat engine is an internal combustion engine.

The heat engine may be a Diesel-cycle engine or, alternatively, an Otto-cycle engine.

A mechanism in accordance with the invention will now be described by way of example only with reference to the accompanying drawings in which:

Fig. 1 represents a front view of the mechanism

showing a guide slot of a first guide member accommodating six pin members and a piston-and-cylinder assembly having six cylinders rotatably mounted on a shaft,

Fig. 2 represents a front view of a second guide member having a guide slot accommodating four pin members,

Fig. 3 represents a front view of a piston-and-cylinder assembly showing details of a cylinder of a four-cylinder arrangement,

Fig. 4 represents a side view of the piston-and-cylinder assembly of Fig. 3,

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Fig. 5 represents a cross-section through the axis of a pin member and a guide member showing bearing means between the pin member and the guide member,

Fig. 6 represents and end view of the pin member supported by a plurality of rollers forming part of the bearing means of Fig. 5,

Fig. 7 represents a front view of a third guide member 15 having a shape of guide slot differing in shape from the first and second guide slots,

Fig. 8 represents a cross-section of the guide member of Fig.7 taken along the line AA,

Fig. 9 represents a cross-section of the guide member 20 of Fig.7 taken along the line BB,

Fig. 10 represents a perspective view of the guide member of Fig. 7 and

Fig. 11 represents another perspective view of the guide member of Fig. 7.

Referring to Fig. 1 of the accompanying drawings, the mechanism includes a piston-and-cylinder assembly including a disc-shaped carrier member 1 on which are mounted six cylinders 2 to 7 housing six pistons 8 to 13. Six pin members 14 to 19 pass diametrically through the pistons 8 to 13 and one set of the ends of the pin members 14 to 19 are accommodated in a guide slot having peripheries 20, 21 in a guide member (not shown). The piston-and-cylinder assembly and the guide slot have a common axis 22. The

piston-and-cylinder assembly is mounted on a shaft 23 the axis of which is the common axis 22. The piston-and-cylinder assembly is attached to the shaft 23 and is rotatable about the common axis 22.

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Although not shown in Fig. 1, the mechanism includes a second guide slot in a second guide member for accommodating the other set of the ends of the pin members 14 to 19, the guide slots being of the same shape.

The six cylinders 2 to 7 are evenly spaced around the disc-shaped carrier member 1, their axes being 60 degrees apart. The peripheries 20, 21 of the guide slot are of a rhomboid shape with curved corners resulting in a guide slot which is of a rhomboid shape with curved corners. The curved corners of the guide slot are parts of ellipses and the guide slot is relatively straight between the curved corners.

In the operation of the mechanism shown in Fig. 1, the piston-and-cylinder assembly rotates about the common axis 22 and the pin members 14 to 19 move along the guide slot, the pistons 8 to 13 sweeping up and down the cylinders 2 to 7 as the pin members 14 to 19 move along the guide slot.

The mechanism shown in Fig. 1 forms a part of a heat engine in which the pistons and cylinders are pistons and cylinders of the heat engine and, in operation, generate motive power for the mechanism.

The details of the heat engine are not shown in Fig. 1
The energy generated in the cylinders 2 to 7 and
pistons 8 to 13 as cylinders and pistons of the heat
engine, causing the pistons 8 to 13 to reciprocate in the
cylinders 2 to 7, is translated into rotary motion of the
piston-and-rotor assembly a s a result of the pin members
14 to 19 moving along the guide slot shown in Fig. 1 and
the second guide slot which is not shown. Poppet valves are

included in the heads of the cylinders 2 to 7 and are controlled by static cams which are adjustable in order to vary the lift and dwell, thereby altering the performance of the heat engine. Co-operating cams are housed in a static part of the heat engine housing and a valve lifting mechanism may be included in the event that it is deemed necessary. In the case of a Diesel-cycle heat engine, an additional cam is added in order to operate an injection pump on the cylinder head or electrically operated injectors may be provided.

The operating gas for the heat engine enters by way of the centre of the shaft 23 and is transferred to the heads of the cylinders 2 to 7 by way of radial ports. The gas charges are compressed in the ports as a result of acceleration due to the rotary motion of the piston-and-cylinder assembly, giving an added benefit of precompression and added density within the combustion chamber controlled by the inlet valve.

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An exhaust port is controlled by an exhaust valve and exhaust gases leave the heat engine by way of a radial gallery on the side of the heat engine. Engine lubrication is effected by a pressurised oil system and cooling is effected by a radiator. Rotation is counter-clockwise as viewed in Fig. 1.

25 The arrangement shown in Fig. 1 is ideal as regards balance since the opposed-piston configuration provides good counterbalance. For good balance, a minimum of two cylinders is required.

Variation of the shapes of the guide slots may be used in order to vary the performance of the mechanism and a heat engine including the mechanism. Variation of the size and, consequently, the weight of the piston-and-cylinder assembly provides for different power characteristics.

There are four parts to the guide slot corresponding to the induction, compression, expansion and exhaust strokes of the operating cycle of a reciprocating heat engine and, consequently, there is ignition once per revolution. That contrasts with more conventional four-stroke arrangements in which there is ignition once every two revolutions, thus a heat engine including the mechanism produces twice as many ignition strokes as a conventional four-stroke arrangement, leading improved power output at lower revolutions.

Referring to Fig. 2 of the accompanying drawings, a guide member 30 having a guide slot with peripheries 31, 32 accommodating four pin members 33 to 36 has a common axis... 37 with a corresponding piston-and-cylinder assembly which is not shown. The corresponding piston-and-cylinder 15 assembly has four pistons and cylinders spaced ninety degrees apart for ideal balance conditions. The operation of the mechanism of Fig. 2 is the same as the operation of the mechanism of Fig. 1 described above and the operation of a heat engine including the mechanism of Fig. 2 is the 20 same as the operation of a heat engine as described above with reference to Fig. 1, except that a four-cylinder heat engine is expected to generate less power than a sixcylinder engine.

The guide slot shown in Fig. 2 has a rhomboid form with curved corners which are parts of ellipses. The guide slot has, in effect, four parts and those parts correspond to the four strokes of a four-stroke heat engine which provide the induction, compression, expansion and exhaust phases of operation. There are alternative shapes for the guide slot. For example, the guide slot may be elliptical in shape or may include three parts which are elliptical and a fourth part which is substantially straight, the

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fourt part corresponding to the expansion phase of operation.

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Referring to Fig. 3 of the accompanying drawings, a piston-and-cylinder assembly includes four cylinders spaced ninety degrees apart and one cylinder 40 includes an axial slot 42 through which a pin member 41 passes, the pin member 41 also passing through the piston which is not visible. The axial slot 42 lies on a diameter of the cylinder 40 and facilitates the assembly of the mechanism by holding the piston in the cylinder in the absence of a guide slot for performing that function. The slot 42 is longer than the stroke of the piston as set by the guide slot and, consequently, the slot 42 does not influence the operation of the mechanism.

Referring to Fig. 4 of the accompanying drawings, the pin member 41 of Fig. 3 protrudes both forward and backwards for engagement with front and rear guide slots (not shown) as does a further pin member 51. Valve members 52 and 53 are also shown in Fig. 4.

The pistons of the arrangements described above may be stepped in diameter, having the smaller or smallest diameter at the position at which the pin member is accommodated.

Referring to Fig. 5 of the accompanying drawings, a pin member such as the pin member 14, say, of Fig. 1 has bearing means including an outer bearing assembly and an inner bearing assembly. The outer bearing assembly includes an outer cylindrical shell 142 and a plurality of cylindrical rollers 1421 supporting the outer cylindrical shell 142 on the pin member 14. The inner bearing assembly includes an inner cylindrical shell 141 and a plurality of cylindrical rollers 1411 supporting the inner cylindrical shell 141 on the pin member 14. The inner cylindrical shell

141 includes a peripheral protrusion and the outer cylindrical shell 142 includes a peripheral depression into which the peripheral protrusion of the inner cylindrical shell 142 fits. The outer and inner bearing assemblies 142 and 141 are retained on the pin member 14 by a retaining clip 155 and the fact that the end diameter of the pin member is reduced in relation to the body of the pin member 14.

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The outer periphery 31 of the guide slot in a guide member 300 is narrower than the inner periphery 32a, 32b of 10 the guide slot in the guide member 300 and the inner periphery has two surfaces 32a and 32b between which there is a step. The step in the inner periphery of the guide slot in the guide member 300 permits the outer cylindrical shell 142 to run clear of surface 32a of the inner 15 periphery while in contact with the outer periphery 310 of the guide slot. The inner cylindrical shell 141 runs along the surface 32b of the inner periphery of the guide slot and runs clear of the outer periphery 310 which does not extend up to the inner cylindrical shell 141. The 20 arrangement shown in Fig. 5 permits the pin member 14 to be guided along the guide slot in contact with the inner and outer peripheries of the guide slot, the inner and outer cylindrical shells rotating in opposite senses.

A ball bearing may be introduced at the interface between the inner and outer cylindrical shells 141 and 142 by providing tracks in the inner and outer cylindrical shells 141 and 142 where they interface and placing a plurality of balls in the tracks.

As shown in Fig. 5, the pin member 14 is a cylindrical tube and includes a first plurality of axial apertures 143 - 148 in the vicinity of the bearing means at the end of the pin member 14 giving access to the interior of the pin

member 14. A second plurality of axial apertures 149 - 154 is provided at the opposite end of the pin member 14 giving access to the interior of the pin member 14. The axial apertures serve to distribute lubricant to the bearing means at the ends of the pin member 14, the lubricant being injected into the interior of the pin member 14 through apertures in the guide member 300.

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end view of the pin member 14 at the bearing means shows that there are sixteen rollers 1421 included in the outer bearing assembly.

Referring to Fig. 7 of the accompanying drawings, the guide member 300 includes a guide slot which is symmetrical about the line AA and, also, about the line BB. The guide slot is dumbbell-shaped, including rounded end portions and narrowing between the rounded end portions. Fig. 7 shows four apertures 301 - 304 in the guide member 300. Lubricant injected through the apertures 301- 304 enters a pin member as it passes the apertures 301 - 304 and is delivered to the bearing means on the pin member through the axial apertures 143 - 154 shown in Fig. 5.

Referring to Figs. 8 and 9 of the accompanying drawings, the apertures 301 - 304 for lubricant are shown as is the narrower outer periphery 310 of the guide slot compared with the total width of the surfaces 32a and 32b making up the inner periphery of the guide slot.

Referring to Figs. 10 and 11 of the accompanying drawings, perspective views of the guide member 300 show the apertures 301 - 304, the outer periphery 310 of the guide slot and the surfaces 32a and 32b making up the inner periphery of the guide slot.

Referring again to Fig. 5, as an alternative to the arrangement shown in which the outer and inner bearing

assemblies 141, 1411, 142, 1421 are so mounted as to rotate about the axis of the pin member 14, the outer bearing assembly 142, 1421 may be so mounted in relation to the mounting of the inner bearing assembly 141, 1411 that the outer cylindrical shell 1421 rotates about an axis which is offset from the axis about which the inner cylindrical shell 141 rotates. An offset in the axes of rotation serves to hold the outer and inner bearing assemblies 141, 1411, 142, 1421 more positively in contact with the surfaces 310 and 32b.

A basic mechanism in accordance with the invention requires one piston and one cylinder with a pin member in engagement with one guide slot and, as is indicated in relation to Fig. 3, an axial slot in the cylinder is not essential to the operation of the mechanism although a slot in the cylinder facilitates assembly.

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The basic mechanism is operable as a part of a heat engine, but if driven, is operable as a pump. Diesel-cycle and Otto-cycle heat engines are suitable and steam engines are also suitable.

Further, either the piston-and-cylinder assembly or the guide member providing the guide slot may be rotatable or both may be rotatable to provide counter-rotating output shafts.

As is made clear above, balanced arrangements based on evenly spaced opposed cylinders are preferred and rotation of the piston-and-cylinder assembly is preferred.

The axial slots in the cylinders could lie on a chord of the cylinder instead of a diameter of the cylinder but the position on the diameter leads to better balance.

CLAIMS

- 1. A mechanism including:
- a piston-and-cylinder assembly including a piston housed in a cylinder,
 - a pin member passing through the piston and

 a guide member having a guide slot accommodating an

 end of the pin member,
- the guide slot being so shaped and orientated in relation to the piston-and-cylinder assembly that a common axis exists between the guide slot and the piston-and-cylinder assembly,
- the guide member and the piston-and-cylinder assembly being so mounted as to be rotatable relative to each other about the common axis and

the guide slot being so shaped as to guide the pin member in a continuous curve, causing the piston to sweep up and down the cylinder, when the guide member and the piston-and-cylinder assembly rotate relative to each other.

2. A mechanism as claimed in claim 1, including axial slots in the cylinder, the pin member engaging the axial slots.

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- 3. A mechanism as claimed in claim 2, wherein the axial slots lie on a diameter of the cylinder.
- 4. A mechanism including:
- a piston-and-cylinder assembly including a first piston housed in a first cylinder and a second piston housed in a second cylinder,

a first pin member passing through the first piston and a second pin member passing through the second piston and

a guide member having a guide slot accommodating an end of the first pin member and an end of the second pin member,

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the guide slot being so shaped and orientated in relation to the piston-and-cylinder assembly that a common . axis exists between the guide slot and the piston-and-cylinder assembly,

the guide member and the piston-and-cylinder assembly being so mounted as to be rotatable relative to each otherabout the common axis and

the guide slot being so shaped as to guide the pin

15 members in a continuous curve, causing the first and second
piston to sweep up and down the cylinder, when the guide
member and the piston-and-cylinder assembly rotate relative
to each other.

- 20 5. A mechanism as claimed in claim 4, wherein the form of the piston-and-cylinder assembly permits the first and second cylinders to lie diametrically opposed to each other.
- 25 6. A mechanism as claimed in claim 5, including at least one further pair of diametrically opposed cylinders on the piston-and-cylinder assembly,

further pistons in the further cylinders and further pin members passing through the pistons and being accommodated in the guide slot.

- 7 A mechanism as claimed in any one of claims 1 to 6, wherein the guide slot is of a rhomboid shape with curved corners.
- 5 8. A mechanism as claimed in claim 7, wherein the curved corners are parts of ellipses.
 - 9. A mechanism as claimed in any one of claims 1 to 6, wherein the guide slot is elliptical in shape.

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- 10. A mechanism as claimed in any one of claims 1 to 6, wherein the guide slot includes three parts which are elliptical in shape and a fourth part which is substantially straight, the straight part occupying a position corresponding to the expansion stroke of the piston or pistons.
- 11. A mechanism as claimed in any one of claims 1 to 6, wherein the guide slot has a dumbell shape including20 curved end portions separated by a narrower waist portion.
- 12. A mechanism as claimed in any one of claims 1 to 11, including a second guide member having a second guide slot accommodating the other end of the pin member or the other ends of the pin members, the second guide slot being so shaped and orientated in relation to the piston-and-cylinder assembly as to share the common axis existing between the first guide slot and the piston-and-cylinder assembly

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13. A mechanism as claimed in any one of claims 4 to 6 and claims 7 to 12 when appended to any one of claims 4 to 6,

including axial slots in the cylinders, the pin members engaging the axial slots.

- 14. A mechanism as claimed in claim 13, wherein the axial
 5 slots lie on diameters of the cylinders.
 - 15. A mechanism as claimed in any one of claims 1 to 14, wherein common axis is the axis of a shaft on which the piston-and-cylinder assembly is rotatably mounted, the remainder of the mechanism being fixed.
 - 16. A mechanism as claimed in any one of claims 1 to 14, wherein the common axis is the axis of a shaft on which the guide member is rotatably mounted or the guide members are rotatably mounted, the piston-and-cylinder assembly being fixed.
 - 17. A mechanism as claimed in any one of claims 1 to 14, wherein the common axis is the axis of a shaft on which the guide member is or the guide members are rotatably mounted and the piston-and-cylinder assembly is rotatably mounted on the shaft.
- 18. A mechanism as claimed in any one of claims 1 to 17,

 25 including bearing means at the end of the pin member or the
 ends of pin members accommodated in a guide slot, for
 effecting rolling contact between the peripheries of the
 guide slot and the end of the pin member or the ends of the
 pin members.

19. A mechanism as claimed in claim 18, wherein the bearing means at the end of the pin member or the ends of the pin members includes an outer bearing assembly

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contacting only the outer periphery of a guide slot and an inner bearing assembly contacting only the inner periphery of the guide slot.

5 20. A mechanism as claimed in claim 19, wherein the outer bearing assembly includes an outer cylindrical shell supported by a plurality of outer rollers on the pin member, the outer cylindrical shell lying in contact with the outer periphery only of the guide slot.

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- 21. A mechanism as claimed in claim 19 or claim 20, wherein the inner bearing assembly includes an inner cylindrical shell supported by a plurality of inner rollers on the pin member, the inner cylindrical shell lying in contact with the inner periphery only of the guide slot.
- 22. A mechanism as claimed in claim 20 or claim 21, wherein the outer and inner bearing assemblies are so mounted that the outer and inner cylindrical shells rotate about the same axis.
- 23. A mechanism as claimed in any one of claims 20 to 22, wherein the outer and inner bearing assemblies are so mounted that the outer and inner cylindrical shells rotate about the axis of the pin member.
- 24. A mechanism as claimed in claim 20 or claim 21, wherein the outer and inner bearing assemblies are so mounted that the outer cylindrical shell rotates about an axis which is offset from the axis about which the inner cylindrical shell rotates.

25. A mechanism as claimed in any one of claims 20 to 23, wherein the bearing means includes a ball bearing between the outer and inner bearing assemblies, the balls of the ball bearing running in tracks in the outer and inner cylindrical shells.

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- 26. A mechanism as claimed in any one of claims 1 to 25, including a guide slot having an inner periphery including a step in its profile for accommodating bearing means at the end of the pin member or the ends of the pin members, the bearing means including an outer bearing assembly contacting only the outer periphery of a guide slot and an inner bearing assembly contacting only the inner periphery.
- 27. A mechanism as claimed in any one of claims 1 to 26, including a guide slot having an outer periphery which is narrower than the inner periphery, bearing means at the end of the pin member or the ends of the pin members including an outer bearing assembly contacting only the narrower outer periphery of a guide slot and an inner bearing assembly contacting only the inner periphery of the guide slot.
- 25 28. A mechanism as claimed in any one of claims 1 to 27, wherein a plurality of apertures are included in the pin member or pin members for receiving and distributing lubricant to the end of the pin member or pin members.
- 30 29. A mechanism as claimed in claim 28, including a guide member having at least one aperture so positioned as to permit the delivery of lubricant through the guide member to the pin member or pin members.

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- 30. A mechanism substantially as herein described with reference to and as shown in the accompanying drawings.
- 5 31. A heat engine including a mechanism as claimed in any one of claims 1 to:30, wherein the pistons and cylinders are pistons and cylinders of the heat engine and, in operation, generate motive power for the mechanism.
- 10 32. A heat engine as claimed in claim 31, which is an internal combustion engine.
 - 33. An engine as claimed in claim 32, which is a Diesel-cycle engine.

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34. An engine as claimed in claim 32, which is an Otto-cycle engine.

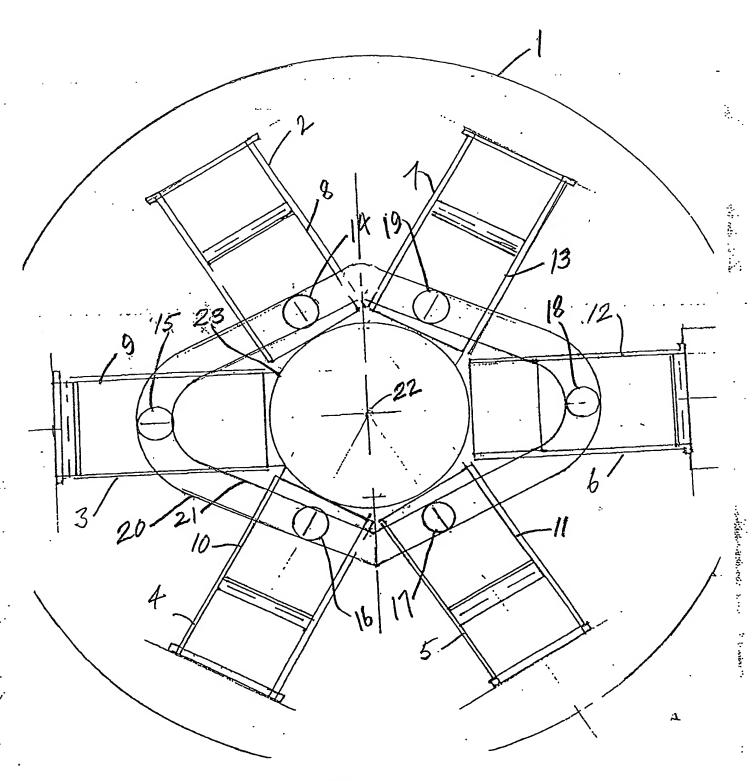
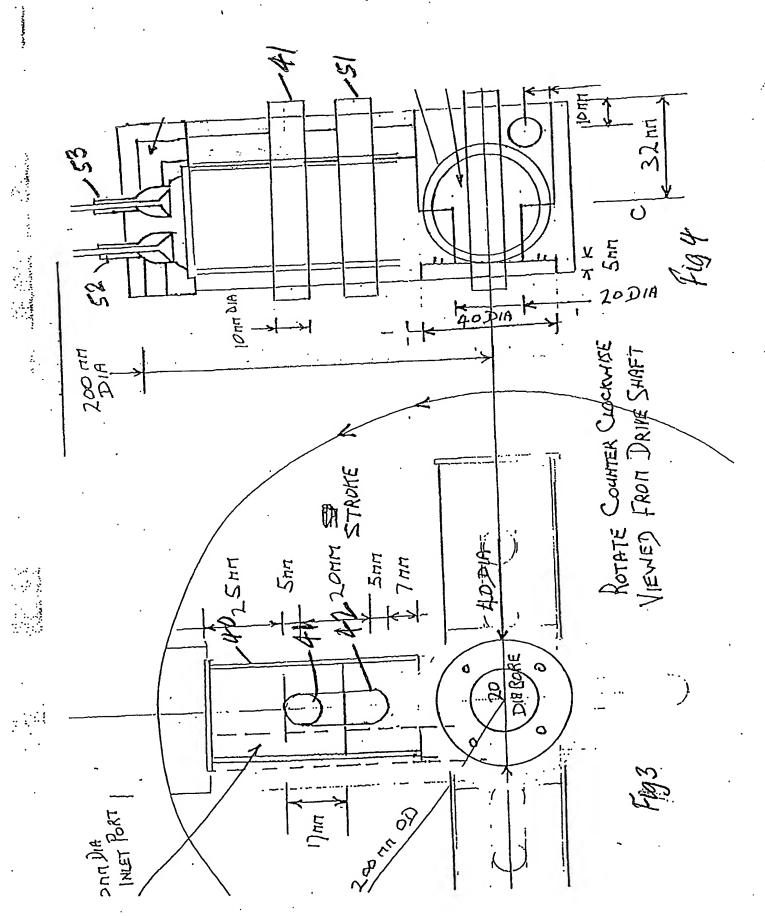


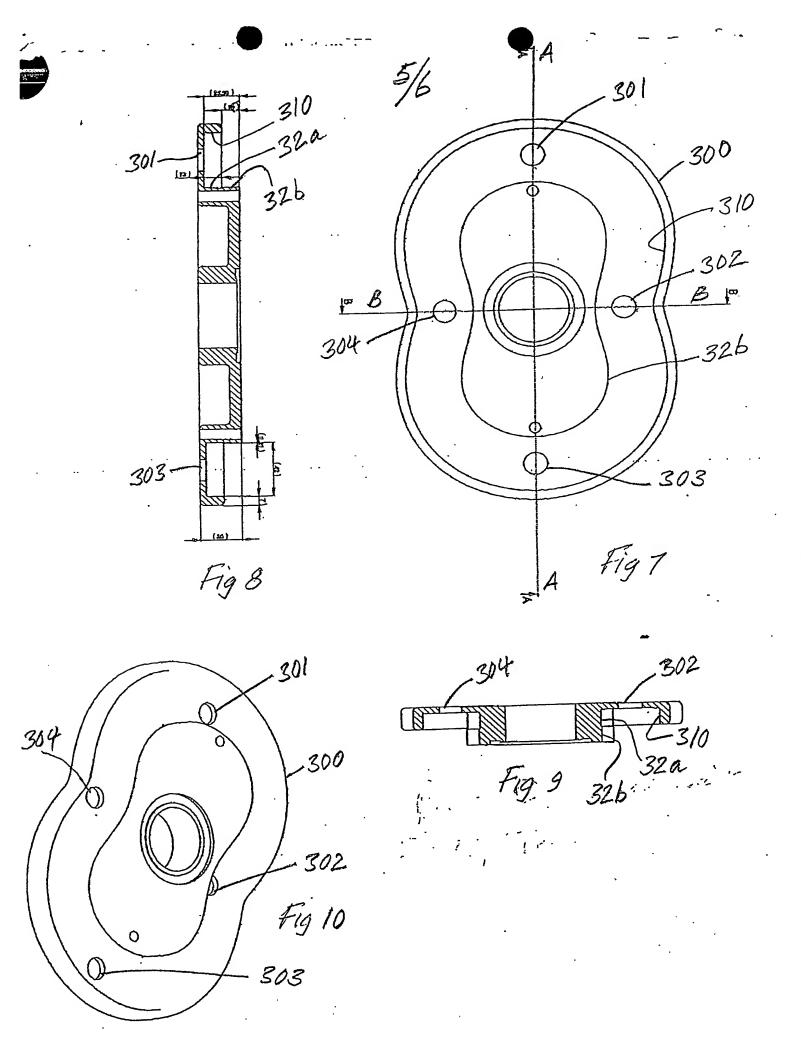
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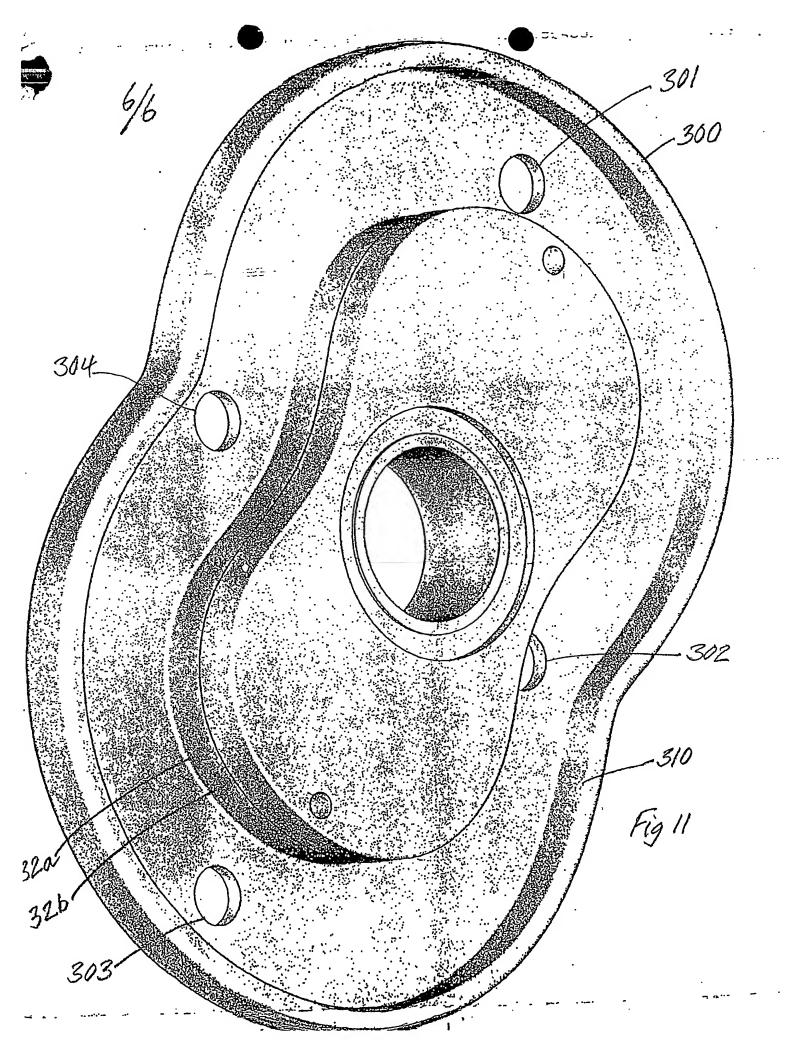
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Fig 2







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